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1	Energy audit of irrigation networks
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16	ABSTRACT
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18	The relationship between water and energy in water distribution systems (WDS) has been a
19	growing concern among energy and water experts. Among the different strategies to improve
20	water-energy efficiency in water distribution networks, energy audits are of paramount
21	importance as they quantify water flow requirements, the amount of energy consumed to meet
22	demand and leakage and friction losses. Previous work has presented the energy audit process
23	for urban WDS and this energy audit is extended to irrigation networks here. This work
24	analyses the most common types of irrigation emitters (sprinklers and pressure compensating

25 and non-pressure compensating drippers), hydrant specifications, irrigation management

26 systems (on-demand or rigid scheduled), and energy losses due to friction in pipes, control 27 valves and irrigation hydrants. The energy audit does not assess whether management of the 28 network is optimal, but analyses the energy consumption. Some of the performance indicators 29 have already been defined for agricultural water networks, some are identical to those of urban 30 WDS, but in addition, a new one is presented that disaggregates the energy dissipated into three 31 terms, energy losses in pipelines, in hydraulic valves and in irrigation hydrants. These indicators 32 show information necessary to better understand the performance of the irrigation network 33 under study, to carry out a deep analysis of energy consumption and to allow for comparison with similar systems. The paper presents the analysis of a real case study conducted on the 34 35 irrigation network of the garden of the Universidad Politécnica de Valencia.

36

37 Keywords: pressurised irrigation, water, energy, audit, leakage, urban irrigation

38

39 1. INTRODUCTION

40 The headline "more crop per drop" perfectly reflects the need for more efficient irrigation, a 41 direct consequence of the substantial increase in irrigated areas in recent decades. To achieve 42 this goal, the strategy has been largely based on converting traditional gravity-fed irrigation into 43 pressurised irrigation systems. And indeed, this has resulted in larger areas being irrigated with 44 the same amount of water. But these water savings have entailed much greater energy 45 consumption, energy itself being a scarce and valuable resource. Table 1 (Corominas, 2010) 46 details water and energy consumption in Spain in the last century and clearly reflects how the 47 situation has changed in a country with a long agricultural tradition.

48

49 Table 1 shows that energy consumption becomes relevant from 1950. The initial increase in 50 energy use cannot be attributed to drip irrigation but the silent revolution (Llamas and Martinez Santos, 2005) which supported the intensive use of groundwater. A couple of decades later, in the 70s, a progressive transformation of irrigation took place from gravity-fed to pressurised irrigation. Table 1 shows that between 1950 and 2007 the irrigated area grew by a factor of 2.5, while water consumption doubled and energy expenditure became 19 times greater.

55

The energy price has been increasing slowly but progressively. This has resulted in a reduction of benefits for farmers. However now prices have risen so much that farmers can no longer sustain this situation and the relationship between water and energy has become a key point on the agenda of developed countries (Department of Energy, 2006). Moreover, the first detailed analysis that quantifies this link between water and energy (CEC, 2005) showed that 19% of the electricity consumption of the State of California was related to water use, a significant amount.

62

On the other hand, although most of this energy consumption occurs in urban and industrial areas, agriculture is also energy hungry. The electricity consumed by agriculture reached more than 4% of the total energy consumed in the state of California (while the water use in agriculture represents 22% of the water consumption of the State). This energy use was divided between water supply (groundwater pumping consumption represented 30% of the total energy consumption in irrigation) and distribution (the remaining 70% was related to water distribution in pressurised irrigation networks).

70

The interest in reducing the energy bill can be addressed using two different and complementary policies. The first (and most natural) strategy deals with the reduction of water consumption, as water savings result in energy savings. This strategy involves a set of actions covered by the term "water demand management". The first step is not to use more water than necessary (in short, to optimise the water delivered to the crop). These needs are directly linked to climatology and to soil moisture. Traditionally, great efforts to quantify the proper amount of
water required in scheduled irrigation have been made. Studies in this area include those related
to climate prediction (WMO, 2010), the use of soil moisture sensors (Greenwood et al., 2010),
deficit irrigation strategies (Geerts and Raes, 2009) and remote sensing and agro-climatic water
balance models (Bastiaanssen et al., 2007; Droogers et al., 2010).

81

82 The second is linked to the optimisation of the design and operation of irrigation networks from 83 an energy-related point of view. This has been an active research area since pressure irrigation 84 began (Allen and Brockway, 1984), and in recent years, for the aforementioned reasons, it has 85 been attracting increased attention. Irrigation networks have to be dimensioned (Farmani et al., 86 2007; Daccache et al., 2009, González -Cebollada et al., 2011) taking into account energetic 87 implications. Furthermore they require pumping stations (Moradi-Jalal et al., 2004; Moradi-Jalal 88 and Karney, 2008; Moreno et al., 2010a) and complementary elements (Kale et al., 2008; 89 Armindo al., 2011) to be implemented to minimise energy expenditure. And once the system is 90 working, its management should also be optimised from the energy perspective (Jimenez-Bello, 91 2010; Lamaddalena and Khila, 2012).

92

93 It should be highlighted that the delivery scheduling method in an irrigation system 94 demonstrates different levels of energy consumption. These schedule types may be classified 95 (Replogle and Gordon, 2007), in order of increasing flexibility, as rigid (rotation, 96 predetermined), central control, intermediate control (arranged) or flexible (on-demand, 97 modifiable). Several studies have shown that between these two extremes, the more flexible the 98 schedule is, the more energy hungry the system becomes (Rodriguez et al., 2009; Moreno et al., 99 2010b). Moreover, other approaches have been carried out to show the influence of 100 management systems on energy consumption in farm systems, considering the life cycle 101 assessment of a crop (Rodrigues et al., 2010), and the energy gain of crops and water

productivity (Chen and Baile, 2009; Guzman et al., 2008). The use of pressurised (or not)
irrigation networks is shown to be a key factor in these analyses.

104

105 Apart from the initial concern over irrigating during the hours at which the electricity tariffs are 106 cheapest (Pulido-Calvo et al., 2003), the requirement of energy optimisation is also considered 107 regarding the design and operation processes of irrigation networks. Moreover, performance 108 indicators of irrigation systems have been defined (Luc et al., 2006; Calejo et al., 2008, Pérez et 109 al., 2009: Moreno et al., 2010c; Rodriguez et al., 2011). And even, in a clear attempt to consider 110 all the possibilities for improvement, the comparison of different systems using benchmarking 111 strategies (Malano and Burton, 2001; Makin et al., 2004; Córcoles et al., 2012) allows the 112 regulator to identify the networks whose practices should be followed.

113

When a decision maker deals with the reduction of energy consumption in irrigation networks, the first step is to properly calculate the amount of water required by crops. The second stage is to quantify the water and energy losses through the network in order to have all relevant information. The two last stages are closely linked as they include showing actions to reduce energy consumption and performing a cost benefit analysis to select the most convenient option.

119

This work deals with the second stage of this process, the quantification of the water and energy consumption in irrigation networks. It includes the use of the energy audit (Cabrera et al., 2010) in agricultural water networks (new terms such as the energy lost in hydraulic valves and hydrants have been added) and the definition of new performance indicators (necessary information to carry out an analysis of energy consumption throughout the system) that consider the key features of irrigation networks.

127 This energy audit is more comprehensive than those that have gone before, including the 128 identification and quantification of all elements that either supply energy to (which can be of 129 two kinds, potential energy supplied by reservoirs, which depends on the height of the header 130 tank or reservoir, and shaft work supplied by the pumps or draw energy from the irrigation 131 network (the energy output is broken down into energy delivered to users (in irrigation 132 networks, this term refers to energy delivered to crops), energy dissipated due to friction and 133 energy losses through leaks (energy lost when water is depressurised and is lost). This last term 134 is not negligible in irrigation networks and its calculation is one of the key objectives of this 135 work. Water losses have always existed in irrigation ditches, although in pressurised water 136 networks they involve energy losses as well.

137

In order to complete the energy audit, two premises should be met. The first is to have calculated the water audit, an easy task if the network has proper metering devices (a flow meter at the head of the network and water meters installed in every irrigated area); while the second is to obtain a calibrated hydraulic model that adheres as closely as possible to reality (unfortunately, all WDS are leaky and the model should consider leaks as pressure-dependent demand when the hydraulic calculations are first done). Once these stages are completed, the energy balance quantifies the amount of energy used for the delivery of water in any network.

145

As commented before, some performance indicators have been defined for agricultural water networks (while those used in urban networks also apply here). These indicators show the information necessary to carry out an analysis of energy consumption throughout the system. The current energy analyses (Moreno et al., 2010c; Rodriguez et al., 2011) are summarised in just one indicator, shaft energy per volume (injected or consumed, kWh m⁻³). The fact that these studies do not disaggregate energy expenditure means that they do not effectively identify or diagnose the weaknesses of the systems they consider. The results obtained with the newperformance indicators show where the head losses are produced.

154

- 155 In conclusion, this work applies the energy audit to a real landscape irrigation network (real case
- 156 study). And according to the values of the indicators, actions to improve water and energy
- 157 management are proposed, the energy benefits are quantified and a cost analysis is performed.

160 Nomenclature

- $C_{e,i}$ Emitter coefficient at node $i \ (m^{(3-\alpha)} s^{-1})$
- C_{si} Emitter discharge coefficient of every sprinkler (m^(3- α) s⁻¹)
- C_1 Context Information Energy nature (dimensionless)
- C_2 Context Information Network energy requirement (dimensionless)
- $E_{dissipated}(t_p)$ Energy losses due to friction for the simulation period (MJ)
- $E_f(t_p)$ Friction energy in pipes for the simulation period (MJ)
- $E_h(t_p)$ Friction energy in hydrants for the simulation period (MJ)
- $E_{input}(t_p)$ Input energy for the simulation period (MJ)
- $E_l(t_p)$ Energy through leaks for the simulation period (MJ)
- $E_{\min,useful}$ Minimum useful energy needed in a frictionless, leak-free network served with the minimum 171 required pressure (MJ)
- $E_{\min,flat}$ Minimum theoretical energy needed in an ideal network, frictionless, leak-free and 173 flat (MJ)
- $E_n(t_n)$ Energy supplied by the reservoirs for the simulation period (MJ)
- $E_{output}(t_p)$ Output energy for the simulation period (MJ)
- $E_p(t_p)$ Energy supplied by pumping stations for the simulation period (MJ)
- $E_u(t_p)$ Energy supplied to users for the simulation period (MJ)

 $E_v(t_p)$ Friction energy in valves for the simulation period (MJ)

- $E_{wasted}(t_p)$ Energy wasted in leakage and dissipation for the simulation period (MJ)
- h_{m_i} Minimum required piezometric head at node *i* (m water column, m.w.c.)
- h_h Head at the sprinklers (m.w.c.)
- $h_i(t_k)$ Piezometric head at node *i* at time t_k (m.w.c.)
- $h_{ni}(t_k)$ Piezometric head at the reservoir *i* at time t_k (m.w.c.)
- $h_{pi}(t_k)$ Piezometric head of the *i* pump at time t_k (m.w.c.)
- h_{si} Piezometric head at the sprinkler j (m.w.c.)
- I_1 Performance indicator excess of supplied energy (dimensionless)
- I_2 Performance indicator network energy efficiency (dimensionless)
- I_3 Performance indicator energy dissipation (dimensionless)
- I_4 Performance indicator leakage energy (dimensionless)
- I_5 Performance indicator standards compliance (dimensionless)
- I_6 Performance indicator characterisation of energy losses (dimensionless)
- I_{61} Performance indicator energy losses in pipes (dimensionless)
- I_{62} Performance indicator energy losses in valves (dimensionless)
- I_{63} Performance indicator energy losses in hydrants (dimensionless)
- *n* Number of demand nodes of the network (dimensionless)

- n_i Number of time intervals ($t_p = n_i \cdot \Delta t$) (dimensionless)
- n_h Number of hydrants of the network (dimensionless)
- n_1 Number of pipes of the network (dimensionless)
- n_n Number of reservoirs (dimensionless)
- n_p Number of pumps (dimensionless)
- n_v Number of valves (dimensionless)
- *m* Number of sprinklers (dimensionless)
- N Rotation speed of the pumping unit using one variable frequency drive (r.p.m.)
- N_0 Nominal rotation speed of the pumping unit (r.p.m.)

205
$$\left(\frac{P_m}{\gamma}\right)_i$$
 Minimum required pressure at node *i* (m.w.c.)

- $q_{hi}(t_k)$ Flow rate at hydrant j at time t_k (m³ s⁻¹)
- $q_j(t_k)$ Flow rate at line j at time t_k (m³ s⁻¹). This term is divided into flow rate that it is consumed

208 and lost through leaks $q_i(t_k) = q_{li}(t_k) + q_{ui}(t_k)$

- $q_{li}(t_k)$ Leakage flow rate at node i at time t_k (m³ s⁻¹)
- $q_{ij}(t_k)$ Flow rate at line j at time t_k (m³ s⁻¹) that finally is lost through leaks
- $q_{ni}(t_k)$ Flow rate supplied by reservoir *i* at time t_k (m³ s⁻¹)
- $q_{pi}(t_k)$ Flow rate supplied by pumping station *i* at time t_k (m³ s⁻¹)

213
$$q_{ui}(t_k)$$
 Consumed flow rate at node *i* at time t_k (m³ s⁻¹)

 $q_{uj}(t_k)$ flow rate necessary to satisfy the users demand that circulates at line j at time t_k (m³ s⁻¹)

- $q_{vj}(t_k)$ Flow rate at value j at time t_k (m³ s⁻¹)
- t_k Time in the steady state simulation (s)
- t_p Total time of simulation (s)
- 218 X Energy lost by friction of the leaking water flow (dimensionless)
- z_i Elevation of node *i* (m)
- $\forall_L(t_p)$ Total leakage volume for the simulation period (m³)
- $\forall_N(t_p)$ Total volume injected for the simulation period (m³)
- $\forall_U(t_p)$ Total volume consumed by users for the simulation period (m³)
- $\Theta_{u,i}(t_p)$ Total demand of node *i* during the simulation period t_p (m³)
- α Emitter exponent (dimensionless)
- γ Specific weight of water (N m⁻³)
- $\Delta h_i(t_k)$ Friction losses in line *j* at time t_k (m.w.c.)
- $\Delta h_{hi}(t_k)$ Friction losses in hydrant j at time t_k (m.w.c.)
- $\Delta h_{v_i}(t_k)$ Friction losses in valve j at time t_k (m.w.c.)
- Δt Time interval of integration ($\Delta t = t_{k+1} t_k$) (s)

231232 2 METHODOLOGY

233 **2.1 Case study**

234 To illustrate the audit procedure, the programmed sprinkling system used for watering the 235 garden of the Universidad Politécnica of Valencia is analysed (figure 1). The irrigation area of 236 this garden has grown through time and new species have been added to the grass meadow 237 (Festuca arundinacea, Pennisetum clandestinum and Poa annua). There are over 50 deciduous, 238 31 evergreen, 16 coniferous, and 13 palm (or similar) tree species and over 20 different shrub 239 species. Nowadays, the plot is divided into hydro-zones which are grouped according to the 240 landscape coefficient method (Costello and Jones, 1999) depending on water needs and crop 241 evapotranspiration values. The reference crop evapotranspiration has been calculated from local 242 weather data using the Penman-Monteith method (Allen et. al., 1998). For the months of 243 greatest water need, and depending on the hydrozone, overall water needs are 1.7 and 3.9 l m⁻² 244 day, corresponding respectively to the water demand of the least and most exposed areas of the 245 garden.

246

Since the irrigation network has been periodically modified, an inventory to characterise the components of the irrigation network has been created. The network irrigates an area of 10.63 ha and consists of 326 nodes, 186 pipes, a water well, two impeller pumps running in parallel and 141 electrovalves upstream of the water discharge outlets, which are the hydrants. The total length of the network is 4.8 km.

252

The hydrants supply the irrigation subunits, which have been designed under the criteria of uniformity of pressure (and consequently flow) at each subunit. This has been reached using a looped network to maintain the same pressure at every subunit. All the subunits are equipped with pop-up emitters (mainly rotating sprinklers and spray sprinklers). Each subunit sprinkler has been identified (according to their brand, model and installation characteristics) in order to obtain the characteristic curve of each from their technical specifications. 260 Groundwater is fed to the system by two identical pumping units (with a characteristic curve described by the equation $h_{pi}(t_k) = -0.155 \cdot (q_{pi}(t_k))^2 - 0.794 \cdot (q_{pi}(t_k)) + 93.55$, where 261 $q_{pi}(t_k)$ and $h_{pi}(t_k)$ are, respectively, the flow rate (m³ s⁻¹) and the head (in metres of water 262 column, m.w.c.; a unit defined as the pressure exerted by a column of water of 1 m in height at 263 4 °C at the standard acceleration of gravity) at time t_k supplied by the pump *i*. The flow and 264 pressure downstream of the pumping station is measured with a Woltmann meter (class B) 265 equipped with a pulse emitter (1 pulse = 100 litres) and a pressure transducer respectively (full 266 267 scale 1 MPa, accuracy $\pm 1\%$).

268

The irrigation management system is based on central system scheduled delivery. This schedule is not as rigid as rotation scheduled delivery (where the irrigation time allocated to each hydrant is not flexible), and it is not nearly as flexible as on-demand delivery scheduling methods (where the flow into the network is random, as is the number of hydrants open at a given time).

273

Some electrovalves are grouped and defined as an irrigation sector. All of them work simultaneously and their operation is remote controlled. The network sectoring has been performed by technicians and gardeners who consider the different hydrozones, the required irrigation time for each subunit and the hours when electricity rates are lower (night). Their key goal is to distribute the flow supplied by the pumps uniformly, considering some other requirements such as the use of the different irrigation areas or the works to maintain the vegetation.

281

282 **2.2 Energy audit of irrigation networks**

This section briefly describes how to estimate the amount of energy used in irrigation networks. The terms used in the energy audit for urban water systems (Cabrera et al., 2010) have been adapted to irrigation networks and the energy dissipated by friction has been divided into energy dissipation in pipes, control valves and irrigation hydrants.

287

In order to perform the analysis in an extended period (t_p , which can take values such as 1 year, 1 month, 1 day, etc.), it is necessary to divide duration time into n_i intervals of time (Δt_k ; 300, 600, 900, 3600 seconds, etc.). Thus, the total energy consumed in the extended period ($t_p = n_i \cdot \Delta t_k$) is obtained from the sum of the energies consumed in each time interval of the steady state simulation.

293

294 **2.2.1** Energy supplied by the reservoir

295 The external energy supplied by reservoirs is:

$$E_n(t_p) = \gamma \cdot \sum_{t_k=0}^{t_k=t_p} \left(\sum_{i=1}^{n_n} q_{ni}(t_k) \cdot h_{ni}(t_k) \right) \cdot \Delta t_k$$
(1)

where γ is the specific weight of water, n_n is the number of reservoirs, $q_{ni}(t_k)$ and $h_{ni}(t_k)$ are, respectively, the flow rate (m³ s⁻¹) and piezometric head (m.w.c.) supplied from each of the water tanks at time t_k , where Δt_k is the time interval (s).

299

300 **2.2.2. Energy supplied by pumping stations**

301 The shaft work supplied by the pumps is:

$$E_p(t_p) = \gamma \cdot \sum_{t_k=0}^{t_k=t_p} \left(\sum_{i=1}^{n_p} q_{pi}(t_k) \cdot h_{pi}(t_k) \right) \cdot \Delta t_k$$
⁽²⁾

302 where $q_{pi}(t_k)$ and $h_{pi}(t_k)$ are respectively the flow rate pumped by the station (m³ s⁻¹) and the 303 pump head (m.w.c.) at time t_k . This calculation needs to be done for the n_p pumping stations that supply shaft work to the system at each discrete time t_k . This energy is water energy and by considering the performance of each pumping unit (an essential parameter for energy optimisation) the electrical equivalent can be calculated. In this paper, and since the focus is on new concepts, these energy losses have not been included in the analysis.

308

309 2.2.3 Energy delivered to users at consumption nodes

310 The energy delivered to users is:

$$E_u(t_p) = \gamma \cdot \sum_{t_k=0}^{t_k=t_p} \left(\sum_{i=1}^n q_{ui}(t_k) \cdot h_i(t_k) \right) \cdot \Delta t_k$$
(3)

311 where *n* is the number of demand nodes of the network, $q_{ui}(t_k)$ and $h_i(t_k)$ are respectively the 312 flow rate delivered to users (m³ s⁻¹) and the piezometric head (m.w.c.) at node *i* and time t_k .

313

314 **2.2.4 Energy through leaks**

Leaks represent energy leaving the system, formally analogous to the energy delivered to usersalthough from the point of view of the audit it is lost energy. This term is:

$$E_l(t_p) = \gamma \cdot \sum_{t_k=0}^{t_k=t_p} \left(\sum_{i=1}^n q_{li}(t_k) \cdot h_i(t_k) \right) \cdot \Delta t_k$$
(4)

with *n* being the number of nodes in the network, $q_{li}(t_k)$ the leaked flow rate (m³ s⁻¹) in the pipes adjacent to node *i* (and therefore associated with this node) at time t_k , and $h_i(t_k)$ is the piezometric head (m.w.c.) at time t_k in the node where the leak $q_{li}(t_k)$ has been concentrated.

320

321 **2.2.5 Friction energy dissipation**

322 The energy dissipated by friction is divided into energy dissipated in pipes, in control valves 323 and in hydrants. As previously mentioned, the latter two of these parameters are specifically introduced to take into account the singularities of the irrigation networks. These elements can be present in urban water networks, but their influence is much lower (from an energetic point of view) than in irrigation networks. For instance, pressure control valves are common in irrigation networks and their energy dissipation becomes an important factor. Similarly, the particular configuration of an irrigation system may also indicate poor energy management, and therefore local hydrant losses can affect overall network performance.

330

331 **2.2.6 Energy dissipation in pipes**

332 The energy dissipated due to friction in pipes is:

$$E_{f}(t_{p}) = \gamma \cdot \sum_{t_{k}=0}^{n_{k}} \left(\sum_{j=1}^{n_{j}} q_{j}(t_{k}) \cdot \Delta h_{j}(t_{k}) \right) \cdot \Delta t_{k}$$
(5)

where n_i is the number of lines of the network, $\Delta h_j(t_k)$ are friction losses (m.w.c.) in line jat time t_k (this term is the difference in piezometric heads between the initial and final nodes), $q_{uj}(t_k)$ and $q_{ij}(t_k)$ are, in line j, the flow rate necessary to satisfy the users demand and the flow rate that finally is lost through leaks, respectively. Therefore, the total flow rate in line j, $q_j(t_k)$, is the sum of the two previous values.

338

339 2.2.7 Energy dissipation in hydraulic valves

340 The energy dissipated in hydraulic valves is:

$$E_{\nu}(t_{p}) = \gamma \cdot \sum_{t_{k}=0}^{t_{k}=t_{p}} \left(\sum_{j=1}^{n_{\nu}} q_{\nu j}(t_{k}) \cdot \Delta h_{\nu j}(t_{k}) \right) \cdot \Delta t_{k}$$
(6)

341 where $q_v(t_k)$ is the flow rate (m³ s⁻¹) flowing through the hydraulic value j at time t_k , n_v is 342 the number of values and $\Delta h_{vj}(t_k)$ is the piezometric head (m.w.c.) lost in the hydraulic value 343 j (calculated as the difference between the upstream and downstream nodes of the value).

344

345 **2.2.8 Energy dissipation in hydrants**

346 The energy dissipated in hydrants is:

$$E_{h}(t_{p}) = \gamma \cdot \sum_{t_{k}=0}^{t_{k}=t_{p}} \left(\sum_{j=1}^{n_{h}} q_{hj}(t_{k}) \cdot \Delta h_{hj}(t_{k}) \right) \cdot \Delta t_{k}$$

$$\tag{7}$$

347 where $q_h(t_k)$ is the flow rate (m³ s⁻¹) flowing through the hydrant *j* at time t_k , n_h is the 348 number of hydrants and $\Delta h_{hj}(t_k)$ is the piezometric head (m.w.c.) lost in the hydrant *j* 349 (individual elements, water meters, filters, valves, etc.).

350

351 **2.3 Final balance**

From the preceding terms, where t_p is the period of calculation of the expressions (commonly one year), the following final balance results:

354

355
$$E_{input}(t_p) = E_n(t_p) + E_p(t_p) = E_u(t_p) + E_l(t_p) + E_f(t_p) + E_v(t_p) + E_h(t_p) =$$

$$= E_{output}(t_p) + E_{dissipated}(t_p) = E_u(t_p) + E_{wasted}(t_p)$$
(8)

Equation (8) states that the energy supplied by reservoirs and pumps to the water coming into the network is equal to the energy delivered to the users (throughout the water supplied) plus the losses (leakage and friction) $E_{wasted}(t_p) = E_l(t_p) + E_f(t_p) + E_v(t_p) + E_h(t_p)$. From this balance, energy losses can be evaluated and efficient actions aimed to improve system'sefficiency can be planned.

362

363 These equations might be solved using water network modelling software to calculate all the 364 values required (flow rates, piezometric head, friction losses, etc. in any element and at any 365 time). The energy audit requires a calibrated model and the water balance, which needs to be 366 calculated in advance. The key to performing the energy audit might be to get all the 367 information from a single network, and this can readily be achieved using data loggers, remote 368 sensors, monitoring devices and information systems such as GIS (Pereira et. al., 2002; Playan 369 and Mateos, 2006; MARM, 2006; Avellá and García-Mollá, 2009). In fact, this situation is 370 increasingly common, and even more so in areas where water is scarce. Once these 371 requirements are met, all these values can be calculated using the water network modelling 372 software, and the equations can be solved. The software selected here has been EPAnet 373 (Rossman 2000), maybe the most widely used around the world. This software is used to calculate the flows, heads, head losses, etc. in all the pipes and at all the nodes in the model. 374 375 EPAnet is demand-driven modelling software that uses temporal demand pattern multipliers to 376 represent a diurnal demand curve, and a 168 h (1 week) extended period simulation may be 377 performed.

378

379

2.4 Tools to assess performance system

380 **2.4.1 Context information and Performance Indicators in irrigation networks**

Context information and Performance Indicators defined elsewhere for water supply systems (Cabrera et al., 2010) are also valid for irrigation networks. In the following paragraphs, their mathematical expressions and a new performance indicator for irrigation networks are presented (table 2). For a better understanding of these indicators, two terms are explained here. The first is the minimum useful energy ($E_{min.useful}$), the energy when delivering the flow at each node 386 from the minimum required head $(h_{mi}=z_i+\left(\frac{P_m}{\gamma}\right)_i)$. The second deals with the theoretical

387 minimum required energy for a flat, leak-free and frictionless network ($E_{\min, flat}$).

388

389 Although all context information and Performance Indicators presented reveal new information, 390 some of them are of paramount importance. The context information will help to identify easily 391 whether these energy analyses are necessary or not; it shows the energy obtained without 392 pumping (C_1) , and if the network is flat or hilly (C_2) . The energy audit will be performed if 393 context information (which can be obtained in the absence of a hydraulic model) recommends 394 it. The most relevant performance indicator is network energy efficiency (I_2) as it represents 395 the portion of energy delivered to crops; this indicates whether the irrigation network is properly 396 managed. Next come energy dissipation (I_3) , characterisation of energy losses (I_6) (both of which refer to design and network sectoring processes) and leakage energy (I_4) (related to 397 398 operation and management issues). Finally, excess of supplied energy (I_1) and standard compliance (I_5) reveal if regrouping of the numerous hydrants can reduce energy expenditure. 399

401 As irrigation networks generally have higher amounts of dissipated energy than urban water 402 systems, an indicator for the determination of energy losses, I_6 , is defined that estimates the 403 importance of dissipated energy divided by the energy expended in the network. This indicator 404 ranges from 0 to 1, where values close to zero indicate that the network is oversized (low 405 friction losses), while values close to 1 indicate leak-free networks. This indicator complements indicators I_3 and I_4 , providing a more detailed analysis of the network. Furthermore, as energy 406 dissipation occurs in pipes, hydraulic valves and hydrants, the indicators I_{61} , I_{62} and I_{63} define 407 their relative importance, where $I_6 = I_{61} + I_{62} + I_{63}$. 408

For any water network, the sum of energy efficiency, dissipated energy and leakage energy takes a value close to and above 1 ($I_2 + I_3 + I_4 = 1+X$). This excess (X) represents the energy lost by friction of the leaking water flow, with values that ranges from 0 (in leak-free networks) to 1 (an ideal and maximum value that would mean that all the input energy is lost by friction of the leaking water flow).

415

416 **2.5 Simulation stage**

417 The main features of the network are:

The irrigation subunits (manifolds, lateral and sprinklers) are installed at the water use
nodes and, although considered in the characterisation of water consumption, for
simplicity they have not been included in the hydraulic simulation model. The flow rate
of the sprinklers depends on the water pressure through the discharge equation.

422
$$q_{hi}(t_k) = \sum_{j=1}^{j=m} C_{sj} \cdot h_{sj}^{\alpha} = \left(\sum_{j=1}^{j=m} C_{sj}\right) \cdot h_h^{\alpha}$$
(9)

423 where C_{sj} (m^(3- α) s⁻¹) is the emitter discharge coefficient assigned to each node of the 424 system to calculate the flow rates of every sprinkler, *m* is the number of sprinklers 425 installed at the garden, α is the exponent of the emitter ($\alpha = 0.5$) and h_{sj} (m.w.c.) 426 represents the piezometric head at the sprinkler *j*. As the pressure at every subunit is 427 constant (as a consequence of the hydraulic design of the subunit, which leads to a 428 suitable diameter of the pipe and a looped network that ensures a constant pressure), 429 the piezometric head at every sprinkler can be expressed as h_h (in m.w.c.).

430 In the simulation model, nodes were grouped into a single characteristic equation that 431 represents all emitters of each subunit. Thus, the head losses at each subunit are 432 assumed to be negligible, which means that the inlet pressure at each rotating or spray433 sprinkler is equal to that existing downstream of the electrovalve.

434 2. The behaviour of each hydrant is simulated by setting a variable pressure drop to each 435 electrovalve. Three diameters (32, 50 and 63 mm) and six different brands are used in 436 the garden (resulting in 15 different types of hydrant). The relationship between 437 pressure drop and flow through the hydrant has been characterised in the laboratory 438 (figure 2 shows an example) and these results have been compared with the 439 information provided by the manufacturer. This requires each hydraulic element and its 440 behaviour to be identified once again in the simulation model. The minimum required 441 pressure at the nodes for the correct operation of the sprinklers takes a value of $\left(\frac{P_m}{\gamma}\right) = 15$ m.w.c. This value has been adopted with regard to technical 442

recommendations and the practical experience of the technicians and gardeners. Atlower values of pressure, the pop-up and proper functioning are not guaranteed.

3. The model also considers leakage. The leaks have been measured using the night-flow 445 446 method (UKWIR, 1994). This method requires the level of leakage to be measured 447 when the delivered water is a minimum (and consequently the pressure is a maximum). 448 Therefore, all the hydrants were closed to measure water consumption (using the 449 Woltmann meter downstream of the pumping station), which in this scenario coincides 450 with leakage. The leaks are assumed to be uniformly distributed (a simplification that 451 comes from the fact that pipes are made of the same material and of the same age in the 452 case study and from the difficulty in finding leaks throughout the system) and are 453 grouped at the nodes in proportion to the length of the converging pipes (Almandoz et 454 al., 2005). The four basic approaches to leakage management are pressure 455 management, active leakage control, speed and quality of repairs and pipes renewal 456 (Lambert and McKenzie, 2002), but leakage management practitioners are well aware that real losses cannot be totally eliminated (OFWAT, 2007) and the volume of 457

unavoidable annual real losses (UARL) represents the lowest technically achievable
annual real losses for a well-maintained system. As a consequence of that, small leaks
with flow rates for sonic detection if non-visible (background leakage) are not
economically viable to repair. Leaks are represented as atmospheric relief valves
(emitter coefficient), at each node of the network (like the water flow consumed). The
design of each emitter has been made according to expression (Rossman, 2000)

464
$$q_{li}(t_k) = C_{e,i} \cdot h_i(t_k)^{\alpha}$$
 (10)

465 where $C_{e,i}$ (m^(3- α) s⁻¹) is the emitter coefficient assigned to each node of the system, 466 $h_i(t_k)$ (m.w.c.) represents pressure drops experienced by the water when passing 467 through the hole and α is the exponent of the emitter. A value of α =0.5 is adopted 468 herein. With the above expression, the leaks in the model are pressure driven demand. 469 Leaks are not typically represented in this way because most hydraulic simulation 470 software, including EPAnet2.0, represents water consumption as independent of 471 pressure (demand driven).

472 4. As commented before, the irrigation management system is based on central system 473 scheduled delivery. This type of operation is similar to the operation of many 474 agricultural farms (and many networks of water user associations), where the modern 475 technologies related to the operation and management of reservoirs, supply systems 476 and hydraulic valves provide the effective use of automation and remote control for 477 systems managers. This feature is considered in the hydraulic simulation model as a 478 control valve (opened when an area has to be watered according to scheduled 479 irrigation). Although the audit is calculated for a period of one year (used as a reference 480 for comparison of results and indicators), irrigation is scheduled weekly. All these 481 features were incorporated in the hydraulic model.

The energy use of on-demand delivery scheduling networks can be calculated using the EPAnet software and running a high number of simulations using the EPAnet toolkit. In each simulation, the total amount of water delivered has to be the same (as the energy consumption is linked to water consumed) and as the irrigation time of the hydrants is a fixed value (i.e. 3.5 hours per day), the opening time of each hydrant would be the parameter that would be modified randomly at each simulation.

489

490 It should also be highlighted that the energy audit performed here (based on the energy equation 491 applied to incompressible fluids) only handles hydraulic equations that do not depend either on 492 final water use or flow regulation. From the energy standpoint, the difference between the 493 delivery scheduling methods is only a matter of boundary and temporal conditions, which are 494 easy to consider using the hydraulic simulation software (with simple or rule-based controls). In 495 the current case study, as the schedule is more rigid, the opening time of every hydrant is 496 determined using control rules in EPAnet (e.g. valve 1.1-A open at time 15:30) and it is only 497 necessary to run one simulation.

498

499 In the model, the water consumed at the end nodes is water used for irrigation while the water 500 consumed at intermediate nodes is leaking water (water losses that do not meet their goal). All 501 the information recovered in the garden is added to the model in the hydraulic simulation 502 software (EPAnet or any other) and the calibration process starts. The objective of the 503 calibration is to observe a good response between the simulated (model predicted) and the 504 observed values (pressures and flows at several points of the network) over the entire simulation 505 period (1 week). The calibration parameters considered here have been the unknown roughness 506 coefficients (the simulations have been carried out using the Darcy-Weisbach equation to 507 calculate the head losses) and the emitter coefficients to achieve better agreement between the 508 observed and modelled pressures and flows respectively (using 5 transducers and one data509 logger). A heuristic process in order to select the location of pressure transducers has been 510 carried out. This process was used to obtain a representative sample of the pressure levels 511 throughout the network for three days in July (when the water demand reaches its maximum).

512

513 **2.6 Scenarios tested**

514 The case study presents the energy analysis of an irrigation system with different leakage rates 515 and with different type of pressure regulation. Case I and Case III represent the current 516 irrigation network, with a volumetric efficiency slightly higher than 75% and a leakage flow rate of 2.166*10⁻⁷ m³ (s⁻¹ m⁻¹) or the equivalent (and more usual 0.78 m³ (km h)⁻¹, which 517 expresses that that every hour, 0.78 m³ are lost in every kilometre of pipe) typical values 518 oscillate between 0.2-2 m³ (km h)⁻¹ in water networks; OFWAT, 2010). Case II and IV 519 520 represent the initial state of the network (or the leak-free situation) with small leakage rates (96% volumetric efficiency and leak rates of 1.66*10⁻⁸ m³ (s⁻¹ m⁻¹), equivalent to 521 $0.06 \text{ m}^3 (\text{km h})^{-1}$). 522

523

524 In Case I and II, pressure regulation is performed using a pressure reducing valve (PRV) (after 525 pumping, the network pressure drops throughout the simulation period to a given value) while 526 in Case III and IV pressure regulation is performed using pumps equipped with variable frequency drive (VFD; figure 3). For hydraulic purposes, the values obtained at the pressure 527 528 transducer (P1, figure 3) located downstream of the pumps (Case III and IV) or downstream of 529 the pressure reducing valve (Case I and II) are the same (55 m.w.c.) in the four cases analysed 530 here. Due to this fact, Cases III and IV show similar hydraulic results to Cases I and II (only the pressure control system has changed). 531

This paper does not intend to demonstrate whether the proposed control system, with two variable speed pumps, is more suitable than other configurations (the regulation problem can be solved with one VFD); the aim of the paper is to show that the pressure regulation systems shown at Cases III and IV are more efficient than the current regulation system.

537

538 **3. RESULTS AND DISCUSSION**

539 **3.1 Results of the water audit**

540 The results of the water audit for the Cases are:

541	•	Input water flow: $\forall_N(t_p) = 4.18 \ 10^{-3} \text{ m}^3 \text{ s}^{-1}$ (equivalent to 0.132 hm ³ year ⁻¹) (Cases I and
542		III) and $3.26 \ 10^{-3} \ m^3 \ s^{-1} \ (0.103 \ hm^3 \ year^{-1})$ (Cases II and IV).
543	•	Delivered water: $\forall_U(t_p) = 3.14 \ 10^{-3} \ \text{m}^3 \ \text{s}^{-1} \ (0.099 \ \text{hm}^3 \ \text{year}^{-1})$ (for all the Cases).
544	•	Real losses: $\forall_L(t_p) = 3.14 \ 10^{-3} \ \text{m}^3 \ \text{s}^{-1} \ (0.033 \ \text{hm}^3 \ \text{year}^{-1})$ (Cases I and III) and
545		0.13 10 ⁻³ m ³ s ⁻¹ (0.004 hm ³ year ⁻¹) (Cases II and IV).
546		

547 **3.2 Results of the energy audit**

The results of the energy audit (MJ consumed per year) are given in table 3. Values in parentheses indicate the percentage that each term represents of the input energy. These values can be converted into MWh (a well known unit for practitioners) dividing them by 3600.

551

The most relevant results in table 3 are the decrease of the energy supplied by pumps in case II (leak-free scenario) in comparison with Case I (real scenario). Energy savings are obtained due to the lower values of energy dissipated (as a consequence of less flow circulating through the network) and due to the decrease in the energy losses through leaks.

Another approach to better energy efficiency in irrigation networks deals with the pressure regulation problem. The head loss at the pressure reducing valve (PRV) in Case I and II is 79-55=24 m.w.c., and the annual energy loss of the valve is 23832MJ (equivalent to 6.62 MWh) (98% of the energy dissipated in the hydraulic valves, table 3). Therefore, the elimination of the control valve and the use of variable speed pumps (Cases III and IV) will improve the energy efficiency of the system as the energy dissipated in hydraulic valves decreases.

563

There are some constraints (VFD price and reliability) that affect the worldwide implementation of this control technique (Pemberton, 2005). The new control system meets the water flow (1-18.5 l s⁻¹) and pressure (55 m.w.c.) requirements of the network through the day. The maximum flow rates (18.5 l s⁻¹) are delivered at a rotation speed equal to $N = 0.9 \cdot N_0$ (where N is the new rotation speed and N₀ is the nominal rotation speed of the pumping unit) while the minimum water flow (1 l s⁻¹) is achieved at a rotation speed equal to $N = 0.77 \cdot N_0$. The energy values for these new cases are given in table 3.

571

572 The comparison of the real scenario data with those after removal of the hydraulic valve (Case I 573 and III, table 3) shows annual energy savings of 111143.9-85212.1=25931.8 MJ (equivalent to 30.87-23.67 = 7.20 MWh year⁻¹). It may seem small, but it represents 23% of the energy spent 574 in the process and it also represents 198MJ (0.055 kWh) for each m³ supplied to the system. 575 576 Considering that the maximum power of the pumps is 12 kW (16.4 HP) and pump speed device 577 (PSD) is supplied with 3-phase 440V, it results in a $3000 \notin$ per PSD (maximum power equal to 578 20 HP). In short, the investment is 6000 \in (as there are two pumps operating in parallel and 579 consequently 2 PSDs are required). Moreover, if considering that the energy costs are 0.15 € kWh⁻¹, the payback period of this alternative is 5.55 years and the annual energy savings 580 581 are 1080 €. This payback period has been obtained only considering the pressure reduction and 582 it will be shortened if other factors are taken into account since the delivery scheduling methods

583 of irrigation also show themselves to be of key importance when obtaining the energy 584 consumption in irrigation networks.

585

586

3.3 Indicator determination and discussion

587 The proposed context information and Performance Indicators provide better insight into the 588 characteristics of the network under study. Their numerical values are depicted at table 4.

589

590 The context-related information has the same values in all the cases (independent of irrigation management mode and leaks in irrigation networks). Energy nature (C_1) ranges from 0 (if all 591 592 the energy is provided by pumps) to 1 (if energy is supplied by the reservoir). This shows that 593 the network is energy hungry and also that further studies on energy reduction are appropriate. 594 Network energy requirements (C_2) range from 1 to infinity, and show whether the network is 595 flat (values close to 1) or hilly (values far from 1). In this case, a value of 1.76, indicating that 596 the network is fairly flat, as there is a 13.7 m difference in height between the highest and 597 lowest nodes of the network.

598

599 By contrast, the Performance Indicators, shown in the remaining columns, depend on the state 600 of the system. In order to clarify the information obtained from the Performance Indicators, the 601 discussion has been disaggregated into the following two sections. The first with regard to 602 Cases I and II (regulation performed with PRV) and the second, Cases III and IV (regulation 603 performed with two variable speed pumps).

605 **3.3.1 Performance Indicators at Cases I and II**

The first indicator (I_1) shows the system's incoming energy with respect to the minimum useful energy, i.e. the outgoing energy at the nodes in the event that all nodes maintain the minimum pressure flow throughout the day. This energy is 4.33 times higher than the minimum useful energy, and if leaks are eliminated, there will be a small improvement of the system, to 3.36 (Case II).

611

About half of the input energy becomes energy delivered to crops ($I_2=0.47$), an acceptable value. The elimination of leaks in the irrigation network (Case II) significantly increases this value ($I_2=0.61$). One third of the total network energy is dissipated by friction ($I_3=0.33$), a typical value in irrigation networks and in Case II, although the value of the annual dissipated energy decreases (31092 compared to 37107.2 MJ; table 3), it is a greater proportion of the input energy, and this indicator slightly increases (Case II, $I_3=0.36$).

618

619 Indicator I_4 quantifies the amount of input energy that is lost due to leakage. In this case study, 620 the total energy lost through leakage is 20% of the overall input energy, a low value. In an ideal 621 leak-free system, its value is 0. The expected values should be between 0.2 and 0.4.

622

The indicator I_5 shows the ratio between the energy delivered to users and the energy when delivering the flow at the minimum required head, and thus it quantifies the additional energy delivered to users (as a consequence of the additional head). The values of this indicator are good and typical of pressurised networks ($I_5 = 2.04$ -2.05; Cases I and II). The closer to one, the better. Therefore, its higher value and better management (Case II) compared to the original network (Case I) reveal that the heads at all the nodes of the network are higher and as a 629 consequence, more energy savings can be obtained following future energy policies. The value 630 obtained here indicates that the network is energetically well-managed (although this is easy as 631 it is a flat network, as indicated by C_2). However, it is always possible to improve energy 632 efficiency, and irrigation indicators serve to identify the major energy losses and potential 633 improvements of the system.

634

The new indicator, I_6 , highlights the amount of energy lost by friction in comparison to the 635 total energy lost in the system. In Case I ($I_6 = 0.63$), the result is typical of networks with leaks, 636 637 while in Case II the value of this indicator is close to one as the network has small leakage rates. Indicators I_{61} , I_{62} and I_{63} reveal that the main energy losses occur in hydraulic valves 638 (I_{62} = 0.41 and 0.56) and that energy losses in hydrants are meaningless (I_{63} = 0, energy 639 640 dissipated in hydrants is low compared to energy wasted). In this particular case study, the 641 energy lost due to friction in hydrants is not relevant, but this indicator can be of paramount 642 importance in other networks.

643

In Case I the energy dissipated by leakage is 0.4% of the input energy $(I_2 + I_3 + I_4 = 1.004;$ X = 0.004). In Case II, leak-free, X is 0.

646

647 The data from the water and energy audits can be used to calculate the energy indicator 648 expressed as kWh m³, which is highly dependent on the topography of the terrain. It has the 649 following values:

• Energy consumed per unit volume injected into the network: 841.62 KJ m⁻³ (0.23 kWh 651 m⁻³) in both cases (which means that 0.23 kWh are consumed to inject one cubic meter of water into the system), a figure that coincides with the estimate made by Corominas
(2010).
Energy consumed per unit volume consumed by the network: 1121.3 and 859.68 KJ m⁻³
(0.31 and 0.24 kWh m⁻³) Case I and II, respectively (values that express that in order to
irrigate one cubic meter of water, 0.31 kWh in Case I and 0.24 kWh are consumed).
These values are greater than the previous one as they consider energy that is lost due to
leaks.

659

660 The results outlined here show that active leakage control in irrigation networks results in 661 energy savings because of the leakage reduction. Beyond these results, cost benefit analysis will 662 describe the economic viability of future actions.

663

664 **3.3.2 Performance Indicators at Cases III and IV**

Cases III and IV show improvements with respect to Cases I and II. The new configuration presents lower energy values ($I_1 = 3.32$ Case III; $I_1 = 4.33$, Case I) (table 4), a greater amount of input energy becomes energy delivered to crops ($I_2 = 0.62$ Case III) and friction losses decrease to very low rates ($I_3 = 0.14$ Case III compared to $I_3 = 0.33$, Case I).

669

The annual energy dissipated in hydraulic valves is now very low in Case III (473 MJ) and although the proportion of energy lost through leakage ($I_4 = 0.20$, Case I; $I_4 = 0.25$, Case III) is larger, this is because the energy input in Case III is much lower than in Case I. To highlight this fact, the annual energy associated with leakage in Case III (20607MJ) is lower than the same term in Case I (21534.2 MJ). The value of standards compliance ($I_5 = 2.04$ -2.05) is the same as before, because the amount of energy supplied does not change. The dissipated energy in comparison to the total energy lost is much lower in Case III than in Case I where most of the dissipated energy is lost in the valve ($I_6 = 0.37$, Case III; $I_6 = 0.63$, Case I). However, in Case III most of the dissipated energy is lost in the pipes ($I_{61} = 0.35$) whereas energy dissipation in hydraulic valves and hydrants is insignificant (table 3).

681

The energy dissipated by leakage is 0.2% of the total input energy $(I_2 + I_3 + I_4 = 1.002;$ X=0.02) in Case III, and 0 in Case IV (leak-free system). The energy consumed per unit volume injected to the network is 0.18 kWh m⁻³ in both cases (a low value, typical of an energetically well-managed network), while the energy consumed per unit volume used is 0.24 and 0.18 kWh m⁻³ (Case III and IV, respectively).

687

688 4 CONCLUSIONS

689 This work has adapted the energy audit, a tool that identifies the end uses of input energy in 690 urban water supply networks, to irrigation networks. The main adjustment has been the 691 decomposition of the energy dissipated by friction into three independent terms: the energy 692 dissipated in pipelines, control valves, and hydrants. This separation allows the decision maker 693 to have more detailed information about the characteristics of the network, and to better identify the primary source of friction losses. A new performance indicator is also proposed for 694 695 highlighting the relevance of energy losses due to dissipation (friction in pipes, valves and 696 hydrants). With this methodology, future actions can be adopted quantitatively (supported by 697 the audit results) and not qualitatively.

699 The key output is a case study to show how the methodology can quantify the energy consumed 700 in irrigation networks and to calculate the energy benefits derived from an efficient management 701 of the irrigation network of the Universidad Politécnica de Valencia. The annual energy savings 702 resulting from the use of a new control system as compared to current operations are (25931.8 MJ, equivalent to 7.20 MWh year⁻¹), a substantial value for a small irrigation system. 703 704 Two case studies (Cases II and IV) have also shown significant annual energy savings in leak-705 free systems (24995 and 18435 MJ), respectively. Therefore, two ways of reducing energy losses in the network under study $(E_{wasted}(t_p))$ have been addressed, namely by reducing 706 707 friction losses and leakage. The adaptation of the energy audit to irrigation systems has proved 708 to be a powerful tool for the development of energy efficient strategies.

709

710 The types of irrigation (on-demand and rigid scheduled), hydrants and emitters (pressure-711 compensating, sprinklers, etc.) have been included (directly or indirectly) in the hydraulic model 712 for the energy analysis of the system. The analysis of their effects is beyond the scope of this 713 study, but they may be relevant when energy losses in irrigation hydrants are substantial. The 714 proper design of the hydrant and irrigation subunit can improve energy efficiency. Additionally, a new indicator (I_{63}) that estimates the energy dissipated in irrigation facilities has been 715 716 defined. The audit also considers leakage in irrigation networks. Water shortage, operational 717 problems, the growing environmental concern and, ultimately, the economic cost of both water 718 and energy losses justify the efforts to prevent leaks in the system.

719

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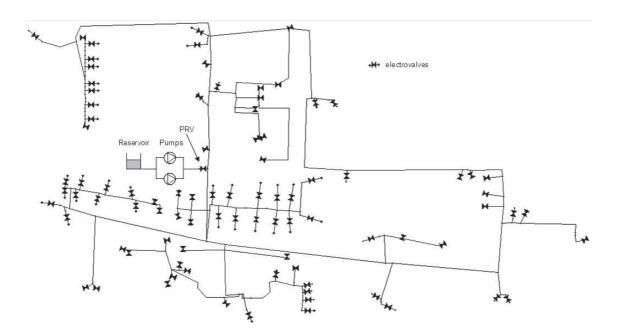


FIGURE 1. SIMPLIFIED LAYOUT OF THE NETWORK

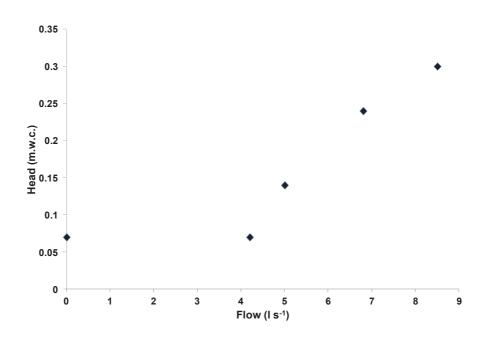
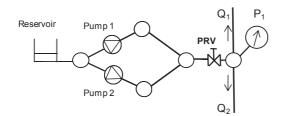


FIGURE 2. EXAMPLE OF WATER LOSS IN A HYDRANT OF THE NETWORK

Pressure regulation in Case I and Case II

Pressure regulation in Case III and Case IV



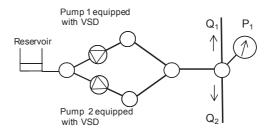


FIGURE 3. TYPES OF PRESSURE REGULATION

Year	Area	Water use	Energy consumption
I cai	(Thousand ha)	(hm ⁻³)	(GWh)
1900	1000	5400	0
1930	1350	7594	182
1940	1500	8288	191
1950	1500	8353	309
1970	2200	12320	1056
1980	2700	14648	2093
1990	3200	17400	3480
2000	3410	18499	4893
2007	3760	20163	5866
Ratios of values in 2007 to the values in			
1950	2.5	2.4	19.0

TABLE 1. EVOLUTION OF WATER AND ENERGY CONSUMPTION IN IRRIGATION SYSTEMS IN SPAIN (COROMINAS, 2010)

			I
C_1	C_2		I_1
Energy nature	Network energy requirement		Excess of supplied energy
$C_1 = \frac{E_n(t_p)}{E_{input}(t_p)}$	$C_2 = \frac{E_{\min,useful}}{E_{\min,flat}}$		$I_1 = \frac{E_{input}(t_p)}{E_{\min,useful}}$
I ₂	i	3	I_4
Network energy efficiency	Energy d	issipation	Leakage energy
$I_2 = \frac{E_u(t_p)}{E_{input}(t_p)}$	$I_{3} = \frac{E_{dissipated}(t_{p})}{E_{input}(t_{p})}$		$I_{4} = \frac{E_{l}(t_{p}) + E_{f}(t_{p}) - E_{f}(t_{p})}{E_{input}(t_{p})}$
I ₅	Ì	6	I ₆₁
Standards compliance	Characterization energy losses		Energy losses in pipes
$I_5 = \frac{E_u(t_p)}{E_{\min,useful}}$	$I_{6} = \frac{E_{dissipated}(t_{p})}{E_{wasted}(t_{p})}$		$I_{61} = \frac{E_f(t_p)}{E_{wasted}(t_p)}$
I 62		I ₆₃	
Energy losses in valves		Energy losses in hydrants	
$I_{62} = \frac{E_v(t_p)}{E_{wasted}(t_p)}$		$I_{63} = \frac{E_h(t_p)}{E_{wasted}(t_p)}$	

TABLE 2. CONTEXT INFORMATION AND ENERGY EFFICIENCY INDICATORS

TABLE 3. ANNUAL ENERGY AUDIT (MJ)

		Case I	Case II	Case III	Case IV
$E_{input}(t_p)$	$E_n(t_p)$	0	0	0	0
	$E_p(t_p)$	111143.9	86148.9	85212.1	66777.0
$E_{input}(t_p)$		111143.9	86148.9	85212.1	66777.0
$E_{output}(t_p)$	$E_u(t_p)$	52414.1 (47.2%)	52526.6 (61.0%)	52414.1 (61.6%)	52526.8 (78.8%)
	$E_l(t_p)$	21534.2 (19.4%)	2442.2 (2.8%)	20607.6 (24.2%)	2336.9 (3.5%)
$E_{output}(t_p)$		73948.4 (66.6%)	54968.8 (63.9%)	73021.8 (85.8%)	54863.7 (82.3%)
$E_{dissipated}(t_p)$	$E_f(t_p)$	12712.3 (11.4%)	12198.1 (14.2%)	11547.4 (13.6%)	11270.8 (16.9%)
	$E_v(t_p)$	24312.8 (21.9%)	18811.7 (21.9%)	473.0 (0.6%)	472.9 (0.7%)
	$E_h(t_p)$	82.2 (0.1%)	82.2 (0.1%)	82.2 (0.1%)	82.2 (0.1%)
$E_{dissipated}(t_p)$		37107.2 (33.4%)	31092.0 (36.1%)	12102.6 (14.2%)	11825.9 (17.7%)